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(54) **ABRADING DEVICE HAVING A FRONT EXHAUST**

SCHLEIFVORRICHTUNG MIT VORDERAUSLASS

DISPOSITIF D'ABRASION À ÉCHAPPEMENT AVANT

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Description

FIELD OF THE INVENTION

[0001] The invention broadly relates to abrading devices, more specifically to pneumatically-powered random orbital devices, and even more particularly to a pneumatically-powered random orbital buffer having a front exhaust.

BACKGROUND OF THE INVENTION

[0002] Random orbital buffing devices are well known in the art. They are used to polish and finish various surfaces without the drawbacks inherent to rotary-type buffing devices. For example, a random orbital buffer may be used to polish a coat of paint on a new automobile. Random orbital buffing devices are commonly pneumatically-powered. After being used to power the device, the compressed air or gas must be exhausted from the device. One problem common to pneumatic devices is that the exhausting air may produce a large amount of noise, which is undesirable for the user of the device.

[0003] For example, abrading tool **10** is shown in Figures 1A and 1B. Tool **10** includes head **12,** which houses a drive means for driving abrasive pad **14.** The drive means may be, for example, a drive means according to United States Patent Nos. 6,206,771 (Lehman) or 4,854,085 (Huber et al.).

[0004] Head **12** is affixed to handle portion **16,** which includes trigger mechanism **18** for controlling the operation of tool **10.** Port **20** is located at the back of the handle portion for coupling the tool to a pneumatic power source, such as a pressurized air tank. Shroud **22** is included to at least partially contain the drive means. Hang ring **24** may be included to provide a convenient means for storing the device when not in use, such as from a hook.

[0005] The published European patent application EP 0 691 181 A1 refers to a pneumatically powered orbital abrading machine having means for cooling an abrading pad and a worked surface. Said cooling means are constituted by an air passage in the driving shaft, from which the air exits through a central hole in the pad and flows outwards, thereby cooling both pad and worked surface as well as removing abraded material.

[0006] Many devices incorporate mufflers to reduce the noise produced by the exhausting air. Traditionally, these mufflers increase the overall size of the device. To reduce the negative effects that this extra size has on the device's usability, these mufflers are commonly placed in or attached to the device's handle, since there is no room to accommodate a muffler in the head portion of the tool proximate the drive means. The channel from the coupling port (port **20**) for the input air is frequently in the handle for the same reason, leading to a common design where the input and exhaust air lines are coaxial or parallel to each other in the handle of the device. That is, separate input and exhaust channels are both included in the handle.

[0007] For example, muffler **26** is included at the rear of tool **10** to muffle the exhaust of the device. This embodiment results in the exhaust air being vented from the rear of the device, near the connector for the input air. This embodiment adds complexity to the device in the form of a second air line that runs the length of the device between the muffler and the outlet of the drive means. Additionally, a constant current of air is exhausted near the user while the device is in use.

[0008] An alternative to this embodiment is included in some grinding devices, which involves venting the exhaust air from the front of the device, onto the abrading pad. Directly exhausting the drive means onto the abrading pad advantageously provides cooling of the pad. Additionally, two separate lines or channels are not required in the handle portion, reducing the complexity of the handle. Also, this eliminates the need to include a muffler, which, in addition to the lack of two channels in the handle, enables more design choices in handle shape and size.

[0009] However, internal space is very limited in the head of these tools, resulting in front-exhaust tools which do not include mufflers. For grinding operations, muffling the exhaust is not a necessity, due to the inherent loudness of grinding. However, muffling is vital for buffing tools to reduce the noise of the tool. Thus, front-exhausting tools tend to be much louder than rear-exhausting tools. Some embodiments attempt to combine the benefits of the front-exhausting and rear-exhausting embodiments by piping the exhaust air from the muffler at the rear of the handle of the device with an exterior line to carry the exhaust back to the front of the device, where it is exhausted onto the pad. This embodiment adds the extra complexity and size for the exterior exhaust line.

[0010] A final problem common to pneumatically-powered buffing devices, and buffing devices generally, is that heat created by the buffing action can damage the surface that is being polished. To prevent the build-up of excess heat, buffing devices are usually limited in speed, or users must operate the devices carefully to ensure particular portions of the surface are not overworked. These limitations reduce the effectiveness of the device, increasing the time needed to polish the surface.

[0011] As can be derived from the variety of devices and methods directed at effectively exhausting pneumatically-powered buffing devices, many means have been contemplated to accomplish the desired end, *i.e.,* preventing the exhausting air from interfering with the buffing action of the device. Heretofore, tradeoffs between noise, device design, preservation of the surface to be polished, and user comfort were required. Thus, there is a long-felt need for a pneumatically-powered buffing device that minimizes exhaust noise and accidental damage to the surface to be polished, while preventing the device's exhaust structures from interfering with the timely and efficient operation of the device.

BRIEF SUMMARY OF THE INVENTION

[0012] The present invention provides an orbital abrading machine as defined in claim 1. The dependent claims define preferred embodiments of the present invention.

[0013] These and other objects and advantages of the present invention will be readily appreciable from the following description of preferred embodiments of the invention and from the accompanying drawings and claims.

BRIEF DESCRIPTION OF THE DRAWINGS

[0014] The nature and mode of operation of the present invention will now be more fully described in the following detailed description of the invention taken with the accompanying drawing figures, in which:

Figure 1A is a side view of a prior art abrading tool;
Figure 1B is a top view of the prior art abrading tool shown in Figure 1;
Figure 2 is a cross-sectional view of a head for an abrading tool according to the current invention;
Figure 3 is an exploded view of the head shown in Figure 2;
Figure 4 is a cross-sectional view of a drive assembly shown in Figure 3;
Figure 5 is an exploded view of the drive assembly shown in Figure 4;
Figure 6 is a perspective view of a front bearing plate of the drive assembly of Figures 4 and 5; and,
Figures 7 and 8 are perspective views of a cylinder of the drive assembly of Figures 4 and 5.

DETAILED DESCRIPTION OF THE INVENTION

[0015] At the outset, it should be appreciated that like drawing numbers on different drawing views identify identical, or functionally similar, structural elements of the invention. While the present invention is described with respect to what is presently considered to be the preferred aspects, it is to be understood that the invention as claimed is not limited to the disclosed aspects.

[0016] Furthermore, it is understood that this invention is not limited to the particular methodology, materials and modifications described and as such may, of course, vary. It is also understood that the terminology used herein is for the purpose of describing particular aspects only, and is not intended to limit the scope of the present invention, which is limited only by the appended claims.

[0017] Unless defined otherwise, all technical and scientific terms used herein have the same meaning as commonly understood to one of ordinary skill in the art to which this invention belongs. It should be appreciated that the term "device" is synonymous with terms such as "tool", "machine", etc., and such terms may be used interchangeably as appearing in the specification and claims. Additionally, the term "buffer," "buffing device," and the like may be used interchangeably. Furthermore, "abrasive pad" or "abrading pad" may be used to refer to any polishing, buffing, abrading, or other pad suitable for such orbital tools. Although any methods, devices or materials similar or equivalent to those described herein can be used in the practice or testing of the invention, the preferred methods, devices, and materials are now described.

[0018] Referring now to the figures, Figures 2 and 3 show buffer head **100.** Head **100** is generally formed by housing **102**, which contains drive assembly **104.** Head **100** is arranged to directly vent the exhaust from drive assembly **104** onto abrasive or buffing pad **105**. Since head **100** is arranged to be held by a user during an abrading operation, grip cover **103** may be included to provide a more comfortable gripping surface for the user. Head **100** may also include hang ring **107,** similar to hang ring **24**, discussed above.

[0019] In the shown embodiment, drive assembly **104** is arranged to enable head **100** to be used for random orbital abrading. For example, drive assembly **104** could generally refer to any suitable drive means for an abrading device, such as taught in the aforementioned '771 or '085 patents, which describe random orbital abrading devices. In the preferred embodiment, drive assembly **104** is regulated by a valve mechanism in a handle portion of a tool. For example, head **100** could affix to any suitable handle known in the art. As a specific example, head **100** could replace head **12** as shown in Figures 1A and 1B, affixing to handle **16**, and powered by a pneumatic source coupled to the buffer via inlet **20** and regulated via trigger mechanism **18** which controls the pneumatic input to the drive means. Advantageously, head **100** would not require rear muffler **26,** as discussed below.

[0020] One embodiment of drive assembly **104** is shown in Figures 4 and 5. In the shown embodiment, the drive assembly comprises rotor **106** having vanes **108.** The rotor and vanes are housed within cylinder **110** between front and rear bearing plates **112** and **114,** respectively. Pin **116** locks the front and rear bearing plates to cylinder **110.** The rotor is rotatable about shaft **118**, with shaft **118** engaged with bearings **120** and **122,** which bearings sit in front and rear bearing plates **112** and **114,** respectively. On one end of shaft **118** is counterbalance **124** for enabling random orbital movement, as described. Bearing **126** is sealed adjacent counterbalance **124** near the end of shaft **118** via v-ring **128** and snap ring **130.** Shaft **132** engages in bearing **126,** and is operatively arranged to connect to a buffer pad, such as buffer pad **105**. Lock ring **134** is provided to secure the drive assembly in housing **102.** Spacer **136** is included to create gap **138** between lock ring **134** and front bearing plate **112.**

[0021] As shown generally in Figures 2 and 3, drive assembly **104** is locked into housing **102** via lock ring **134.** In the shown embodiment, lock ring **134** threadingly engages with interior threading on housing **102** for locking drive assembly **104** in housing **102.** A shroud is

formed by inner and outer shroud portions **140** and **142,** engaged with housing **102.** The inner and outer shroud portions form a shroud chamber **141.** The shroud generally surrounds counterbalance **124** and second shaft **132** near the end of shaft **118.** Outer shroud portion **140** engages with o-ring **144** against lip **146** of housing **102,** and inner shroud portion **142** engages with o-ring **148** against lip **150** of housing **102.** It should be appreciated that the o-rings could be replaced by any other suitable sealing means known in the art for preventing leakage of the exhaust as it travels through head **100.** A least one aperture **152** is included between lips **146** and **150**. In the shown embodiment, aperture **152** is included in groove **154,** between the lips. Groove **154** enables o-ring **144** to expand as the o-ring is moved into engagement with lip **146,** without risk of damaging the o-ring. That is, if groove **154** were not formed between the lips, then a portion of o-ring **144** would likely expand into aperture **152** as the o-ring passes over the aperture, and this portion would like be clipped or sheared off as the o-ring is forced into final engagement with the housing. Since the o-rings prevent leakage of air as it is exhausted out the front of the buffer, it is important that the o-rings are not damaged during assembly.

[0022] Muffling material **156** is included between the inner and outer shroud portions. In one embodiment, the muffling material is a strip of felt. By including muffling material **156** in the gap formed between the inner and outer shroud portions, the shroud effectively acts as a muffler for the buffer. Previously, as discussed above, muffler were included at the far opposite end of the handle from the buffer head, and the handle accordingly required two sealed channels so that the handle could both receive the pneumatic input and expel the exhaust. Thus, if head **100** is utilized, a muffler is not required at the opposite end of the buffing tool. For example, muffler **26** would not be required in tool **10** if head **12** were replaced with head **100.** Additionally, since only one chamber is required in the handle, the arrangement of the handle can be greatly simplified.

[0023] Front bearing plate **112** is shown in more detail in Figure 6. Plate **112** includes annular projection **158,** in which bearing **120** is to be seated. Shaft **118** is insertable through bore **160** for rotatable engagement with bearing **120.** Plate **112** also includes cut **162** in flange **163.** Cylinder **110,** shown in more detail in Figures 7 and 8, includes cut **164** which corresponds to cut **162** in front edge or rim **166.** Conversely, the opposite rim, rear rim **168** provides a constant diameter about the cylinder and does not include a cut. During operation of a tool including head **100,** a pneumatic input (*e.g.*, pressurized air) is fed into drive assembly **104,** which is housed within cylinder **110,** via inlet **170.** The air is exhausted through outlets **172.** Dividing area **174** is at a common diameter with rear rim **168** and the uncut portion of front rim **166** for separating the inlet from the outlet (a similar dividing area is included on the opposite side of the Figures, hidden from view). That is, the housing preferably has an inner diameter which corresponds to the outer diameter of the cylinder for sealing the pneumatic input between rims **166** and **168** in recessed area **176** proximate inlet **170.** The exhaust is expelled from outlets **172** into recessed area **178,** which is bounded on one side by rim **168.** Recessed area **178** generally defines an exhaust cavity between housing **102,** the body of cylinder **110,** and rim **168.** The exhaust is free to exit the housing via cut **164** in front rim **166.** Pin **116** is insertable through bore **180** for engagement with a corresponding bore in rear bearing plate **114,** and partial bore in plate **112** (hidden from view in Figure 4).

[0024] The assembly of head **100** can be best appreciated by referring again to Figures 2 and 3. Grip cover **103** engages over housing **102.** Hang ring **107** clips onto the housing and is held in place due to lip **146.** O-ring **144** seals outer portion **140** of the shroud against lip **146** of the housing. Muffling material **156** is engaged between outer portion **140** and inner portion **142** of the shroud. O-ring **148** seals inner portion **142** of the shroud against lip **150** of the housing, containing muffling material **156** in chamber **141** formed between the outer and inner portions of the shroud. Orifice **152** is included to provide pneumatic communication between cavity **178** and chamber **141** for enabling the exhaust to flow from the cavity to the chamber. Screws **182** secure inner shroud portion **142** to housing **102** via bores **184.** In addition to friction between outer shroud portion **140** and housing **102,** the outer shroud portion is also supported by projections **143** of inner shroud portion **142.** Lock ring **134** is included to lock the top portion of drive assembly **104** within housing **102,** with the bottom portion of the drive assembly surrounded by the shroud. Abrasive pad **105** secures to shaft **132,** which is freely rotatable about a second axis, assisted by bearing **126.**

[0025] Thus, it can be seen that a path can be traced throughout head **100** which enables the exhaust to be expelled directly on the abrasive pad. Specifically, air or some other operating fluid is supplied to head **100** via a port in a handle, such as port **20** in handle **16.** The operating fluid then powers the rotor to rotate drive assembly **104** about shaft **118.** The operating fluid is exhausted via outlets **172** into exhaust cavity **178** between cylinder **110** and the interior of housing **102**. Cuts **162** and **164** enable the exhaust to flow out of exhaust cavity **178** and into shroud chamber **141**. Specifically, in the shown embodiment, spacer **136** between lock ring **134** and plate **112** creates gap **138,** which aligns with holes **152** in housing **102**. Holes **152** align with outer and inner shroud portions **140** and **142** so that the exhaust enters shroud cavity **141**. That is, the exhaust flows through the channel created by cuts **162** and **164** into gap **138**, and from gap **138** through holes **152** into chamber **141**. O-rings **144** and **148** seal above and below holes **152** to prevent leakage of the exhaust. The exhaust then exits shroud chamber **141** via holes **186** in the inner shroud portion and through slots **145** formed between projections **143** and the outer shroud portion.

[0026] Accordingly, the exhaust is directly vented onto the abrasive pad for improved cooling of the pad during operation. By directly, it is meant that the exhaust is contained in the head and must only travel through the head, and not back through the handle. Advantageously, this enables increased buffing speed and buffer pad lifespan, decreased buffing time and a reduced occurrence of imperfections caused on the buffing surface due to overheating of the pad. The shown arrangement also reduces the required complexity of a handle for a tool using head **100,** since the exhaust no longer needs to travel back through the handle, eliminating the need for a rear muffler (*e.g.* muffler **26**). Thus, the above described embodiment enables the shroud to not only protect and contain the rotating components of the drive assembly (counterbalance **124** particularly), but to also muffle the exhaust as it passes through the head to cool the buffing pad.

[0027] Thus, it is seen that the objects of the present invention are efficiently obtained, although modifications and changes to the invention should be readily apparent to those having ordinary skill in the art, which modifications are limited only by the appended claim. It also is understood that the foregoing description is illustrative of the present invention and should not be considered as limiting. Therefore, other embodiments of the present invention are possible within the scope of the present invention as defined by the appended claims.

Claims

1. An orbital abrading machine (10) comprising a head (100) comprising:

   a housing (102);
   a shroud including inner and outer portions (140, 142), defining a shroud chamber (141) between said inner and outer portions (140, 142);
   a drive means (104) for driving an abrading pad (105), said drive means (104) at least partially enclosed by said housing (102) and said shroud, said drive means (104) having a pneumatically-powered rotor, wherein said drive means (104) produces an exhaust which is directly vented into said chamber (141) without leaving said head (100);
   an exhaust cavity (178) for receiving said exhaust from said drive means (104), wherein said exhaust cavity (178) is in pneumatic communication with said chamber (141) for enabling said exhaust to flow from said exhaust cavity (178) and into said chamber (141);
   wherein said chamber includes at least one opening for directing said exhaust toward said abrading pad (105) for cooling said pad (105) with said exhaust, the at least one opening constituted by at least one hole (186) in the inner shroud portion (142);
   wherein said inner and outer shroud portions (140, 142) are engaged against said housing (102) about an orifice (152) with a first seal (144) and a second seal (148), respectively; and
   wherein said orifice provides said pneumatic communication between said exhaust cavity and said chamber, said first and second seals for preventing leakage of said exhaust as said exhaust flows from said exhaust cavity through said orifice into said chamber.

2. The orbital abrading machine (10) of claim 1, wherein said drive means (104) receives a pneumatic input, said pneumatic input sealed from said exhaust except for a path through said drive means (104).

3. The orbital abrading machine (10) of claim 1 or 2, wherein muffling material (156) is contained within said chamber (141) for muffling said exhaust.

4. The orbital abrading machine (10) of one of the claims 1 to 3, wherein said drive means (104) is secured at least partially within said housing (102) with a lock ring (134), wherein a spacer (136) is provided with said lock ring (134) for creating a gap (138), said gap (138) enabling pneumatic communication between said exhaust cavity (178) and said chamber (141).

5. The orbital abrading machine (10) of one of the claims 1 to 4 wherein a handle (16) is secured to said head (100) and said handle (16) including a port (20) for coupling said abrading machine (10) to a source for powering said drive means (104).

6. The orbital abrading machine (10) of one of the claims 1 to 5, wherein the orbital abrading machine (10) which has the head (100) is a random orbital buffer.

Patentansprüche

1. Orbitalschleifmaschine (10), umfassend einen Kopf (100), umfassend:

   ein Gehäuse (102);
   einen Kragen, der innere und äußere Abschnitte (140, 142) aufweist und eine Kragenkammer (141) zwischen den inneren und äußeren Abschnitte (140, 142) definiert;
   ein Antriebsmittel (104) zum Antreiben eines Schleifpads (105), wobei das Antriebsmittel (104) wenigstens teilweise von dem Gehäuse (102) und dem Kragen umschlossen ist, wobei das Antriebsmittel (104) einen pneumatisch betriebenen Rotor aufweist, wobei das Antriebsmittel (104) einen Ausstoß erzeugt, der direkt in

die Kammer (141) entlüftet wird, ohne den Kopf (100) zu verlassen;
einen Ausstoßhohlraum (178) zur Aufnahme des Ausstoßes aus dem Antriebsmittel (104), wobei der Ausstoßhohlraum (178) in pneumatischer Verbindung mit der Kammer (141) steht, um es dem Ausstoß zu ermöglichen, aus dem Ausstoßhohlraum (178) und in die Kammer (141) zu strömen;
wobei die Kammer wenigstens eine Öffnung umfasst, um den Ausstoß in Richtung auf das Schleifpad (105) zu lenken, um das Pad (105) mit dem Ausstoß zu kühlen, wobei die wenigstens eine Öffnung durch wenigstens ein Loch (186) im inneren Kragenabschnitt (142) gebildet ist;
wobei die inneren und äußeren Kragenabschnitte (140, 142) jeweils mit einer ersten Dichtung (144) und einer zweiten Dichtung (148) um eine Mündung (152) herum an das Gehäuse (102) angefügt sind; und
wobei die Mündung die pneumatische Verbindung zwischen dem Ausstoßhohlraum und der Kammer bereit stellt, wobei die ersten und zweiten Dichtungen vorgesehen sind, um ein Entweichen des Ausstoßes zu verhindern, wenn der Ausstoß vom Ausstoßhohlraum durch die Mündung in die Kammer strömt.

2. Orbitalschleifmaschine (10) nach Anspruch 1, wobei das Antriebsmittel (104) eine pneumatische Eingabe aufnimmt, wobei die pneumatische Eingabe mit Ausnahme eines Wegs durch das Antriebsmittel (104) bezüglich des Ausstoßes abgedichtet ist.

3. Orbitalschleifmaschine (10) nach Anspruch 1 oder 2, wobei schalldämpfendes Material (156) in der Kammer (141) zum Schalldämpfen des Ausstoßes enthalten ist.

4. Orbitalschleifmaschine (10) nach einem der Ansprüche 1 bis 3, wobei das Antriebsmittel (104) wenigstens teilweise im Inneren des Gehäuses (102) durch einen Sicherungsring (134) befestigt ist, wobei ein Abstandselement (136) mit dem Sicherungsring (134) vorgesehen ist, um einen Zwischenraum (138) zu bilden, wobei der Zwischenraum (138) eine pneumatische Verbindung zwischen dem Ausstoßhohlraum (178) und der Kammer (141) ermöglicht.

5. Orbitalschleifmaschine (10) nach einem der Ansprüche 1 bis 4, wobei ein Griff (16) an dem Kopf (100) befestigt ist, und wobei der Griff (16) einen Anschluss (20) aufweist, um die Schleifmaschine (10) mit einer Quelle zur Energieversorgung des Antriebsmittels (104) zu verbinden.

6. Orbitalschleifmaschine (10) nach einem der Ansprüche 1 bis 5, wobei die Orbitalschleifmaschine (10), welche den Kopf (100) aufweist, ein Exzenterpolierer ist.

Revendications

1. Machine abrasive orbitale (10) comprenant une tête (100) comprenant :

un boîtier (102) ;
un carénage comprenant des parties intérieure et extérieure (140, 142) qui définissent une chambre de carénage (141) entre lesdites parties intérieure et extérieure (140, 142) ;
un moyen d'entraînement (104) destiné à entraîner un tampon abrasif (105), ledit moyen d'entraînement (104) étant au moins partiellement renfermé dans ledit boîtier (102) et ledit carénage, ledit moyen d'entraînement (104) possédant un rotor à entraînement pneumatique, ledit moyen d'entraînement (104) produisant un échappement qui est directement évacué dans ladite chambre (141) sans sortir de ladite tête (100) ;
une cavité d'échappement (178) destinée à recevoir ledit échappement desdits moyen d'entraînement (104), ladite cavité d'échappement (178) étant en communication pneumatique avec ladite chambre (141) afin de permettre audit échappement de circuler hors de ladite cavité d'échappement (178) et dans ladite chambre (141);
dans laquelle ladite chambre comprend au moins une ouverture pour diriger ledit échappement vers ledit tampon abrasif (105) afin de refroidir ledit tampon (105) avec ledit échappement, l'au moins une ouverture étant formée par au moins un trou (186) dans la partie de carénage intérieure (142) ;
dans laquelle lesdites parties intérieure et extérieure du carénage (140, 142) sont en prise contre ledit boîtier (102) autour d'un orifice (152) muni d'un premier joint (144) et d'un second joint (148), respectivement; et
dans laquelle ledit orifice assure ladite communication pneumatique entre ladite cavité d'échappement et ladite chambre, lesdits premier et second joints étant destinés à empêcher la fuite dudit échappement lorsque ledit échappement circule de ladite cavité d'échappement vers ladite chambre à travers ledit orifice.

2. Machine abrasive orbitale (10) selon la revendication 1, dans laquelle ledit moyen d'entraînement (104) reçoit une entrée pneumatique, laquelle entrée pneumatique est isolée dudit échappement à l'exception d'un trajet traversant ledit moyen d'entraîne-

ment (104).

3. Machine abrasive orbitale (10) selon la revendication 1 ou 2, dans laquelle du matériau insonorisant (156) est contenu dans ladite chambre (141) pour assourdir ledit échappement.

4. Machine abrasive orbitale (10) selon l'une des revendications 1 à 3, dans laquelle ledit moyen d'entraînement (104) est fixé au moins partiellement audit boîtier (102) avec une bague de verrouillage (134), une rondelle d'écartement (136) étant prévue avec ladite bague de verrouillage (134) pour créer un interstice (138), lequel interstice (138) permet la communication pneumatique entre ladite cavité d'échappement (178) et ladite chambre (141).

5. Machine abrasive orbitale (10) selon l'une des revendications 1 à 4, dans laquelle une poignée (16) est fixée à ladite tête (100) et ladite poignée (16) comprend un raccord (20) pour coupler ladite machine abrasive (10) à une source d'énergie alimentant ledit moyen d'entraînement (104).

6. Machine abrasive orbitale (10) selon l'une des revendications 1 à 5, dans laquelle la machine abrasive orbitale (10) possédant la tête (100) est une ponceuse orbitale à mouvement aléatoire.

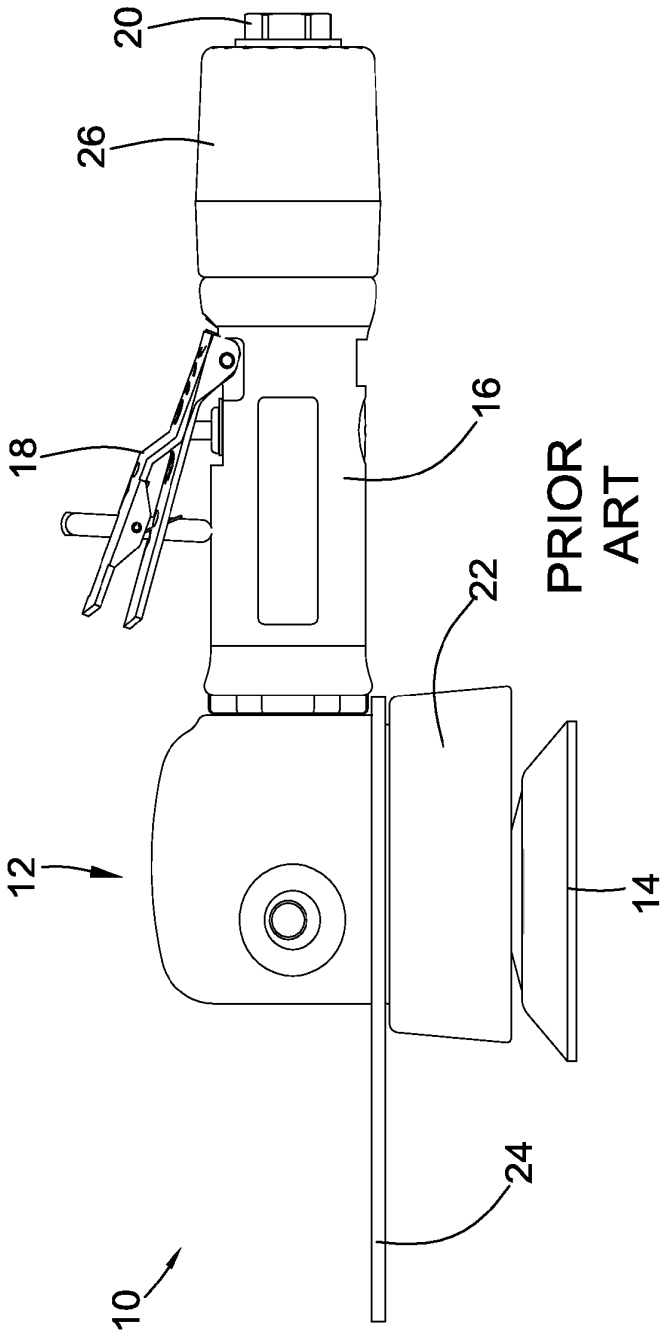
20
26
18
16
22
12
14
24
10
PRIOR ART

Fig. 1A

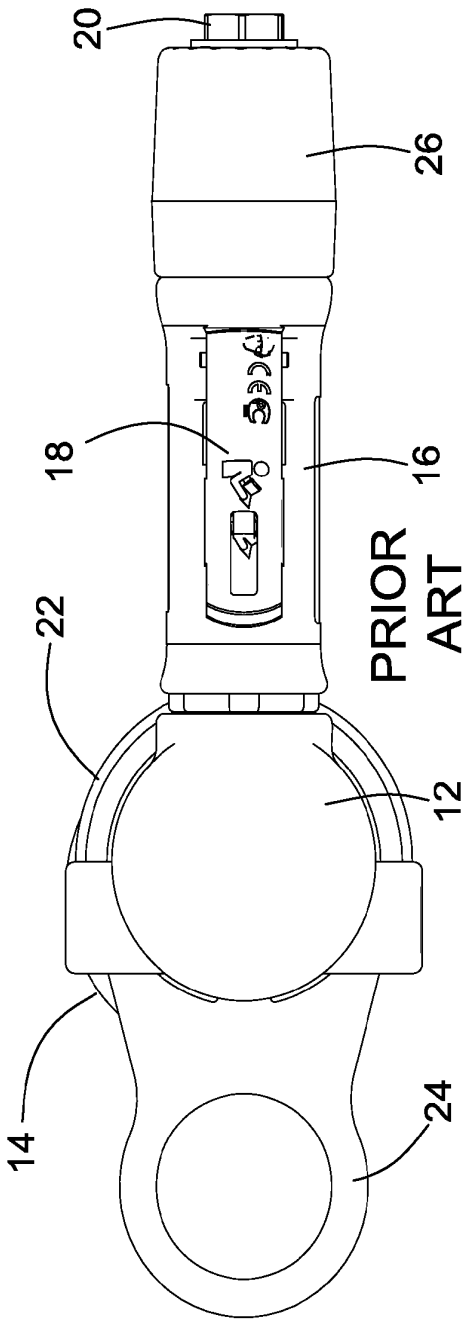
20
26
18
16
22
12
14
24
10
PRIOR ART

Fig. 1B

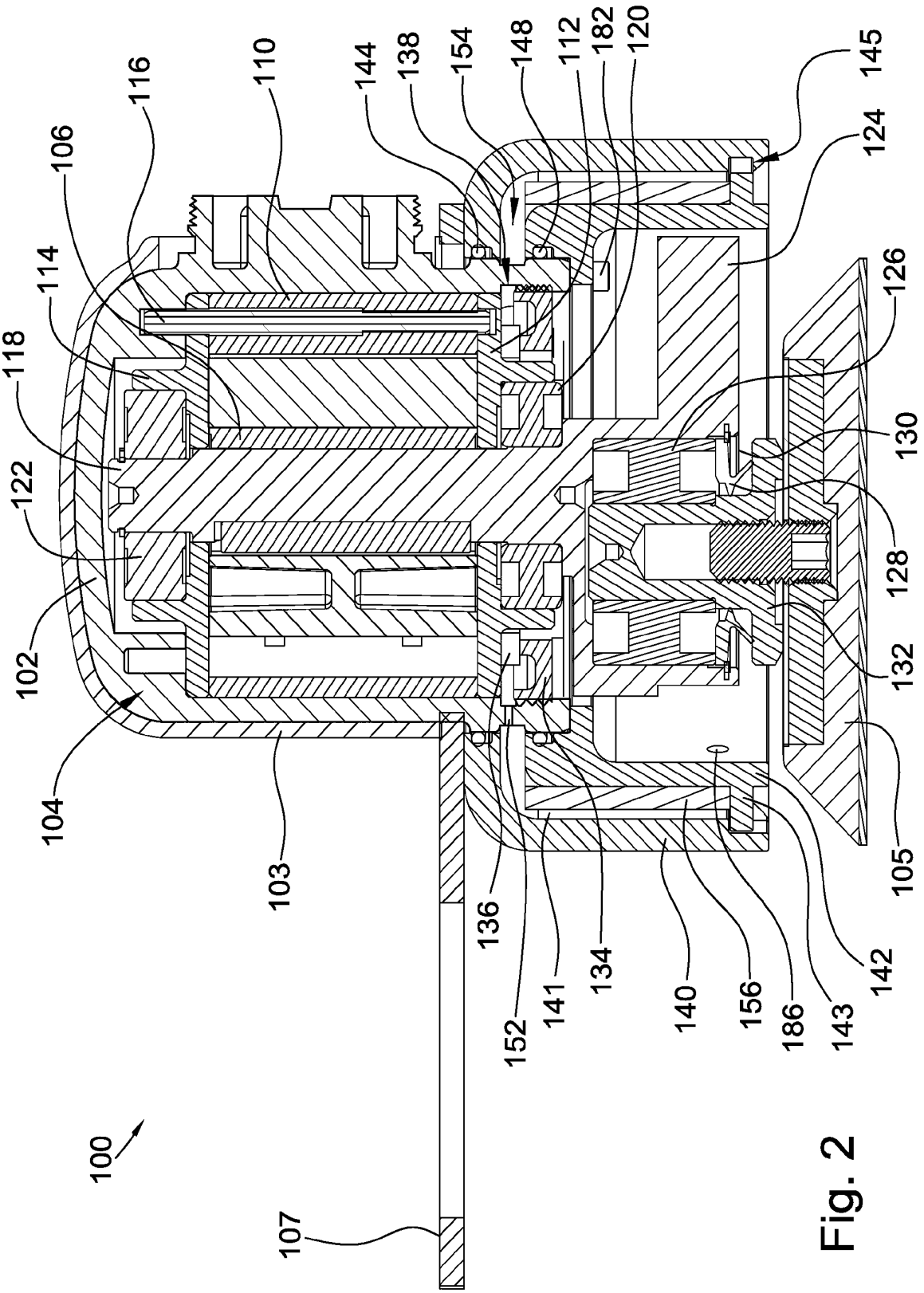
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Fig. 2

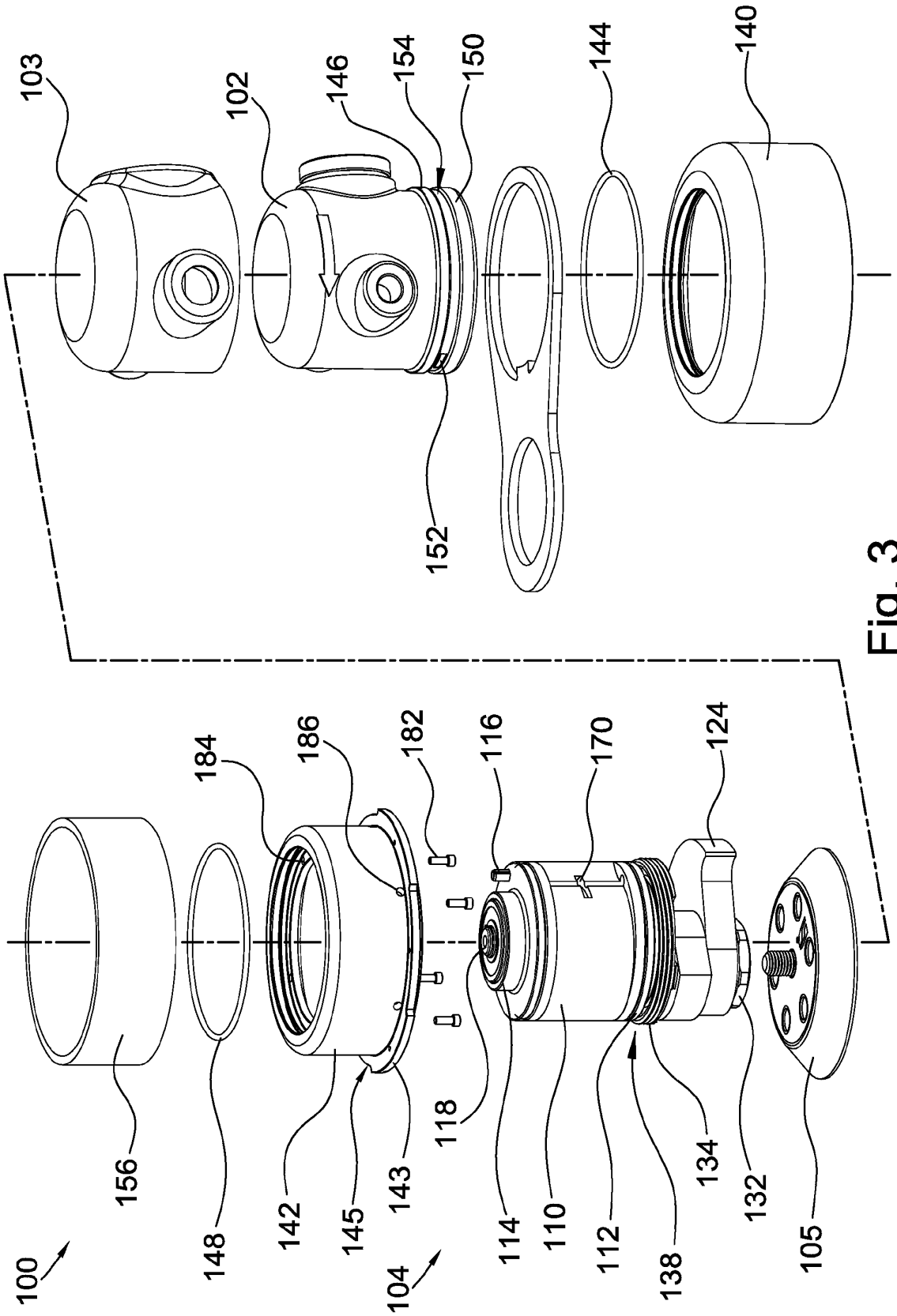
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Fig. 3

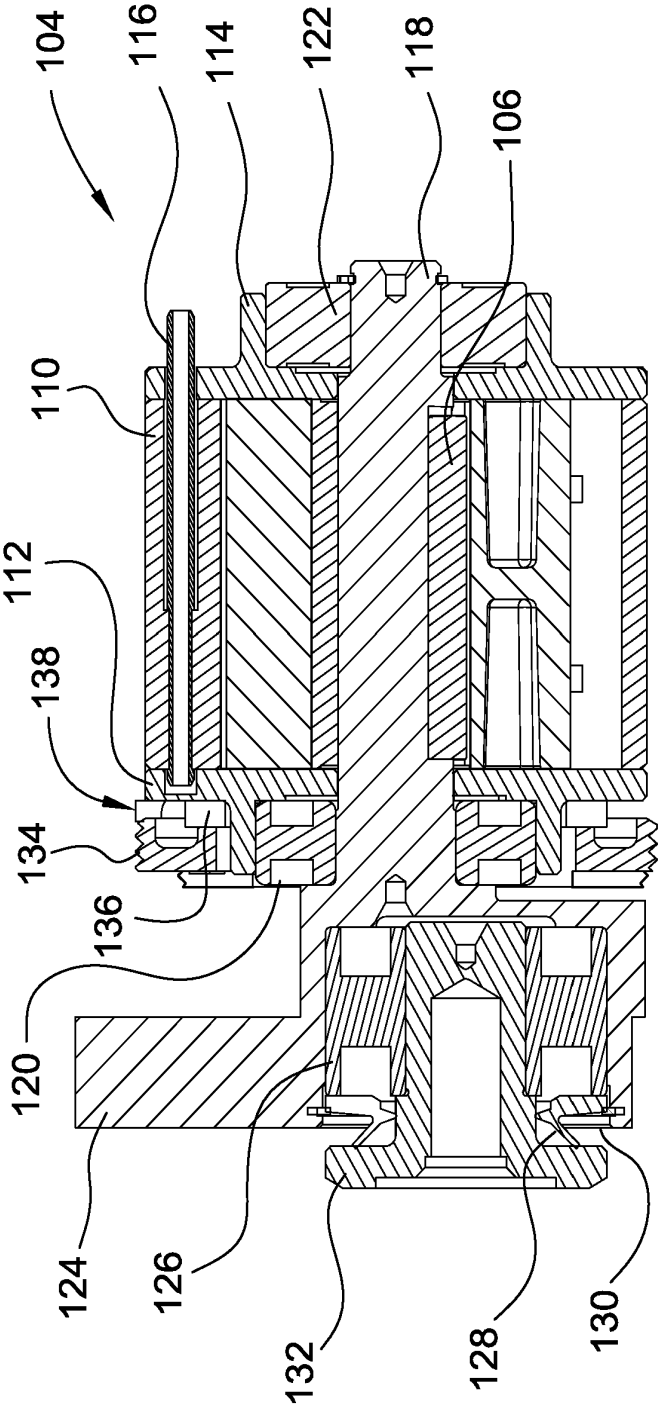
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Fig. 4

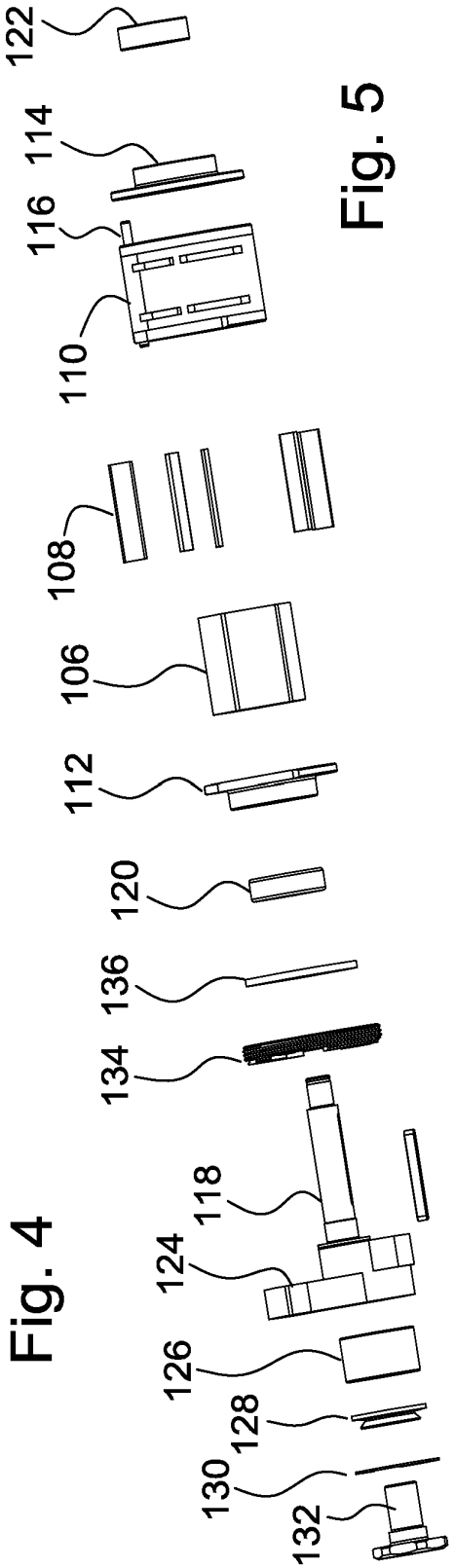
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Fig. 5

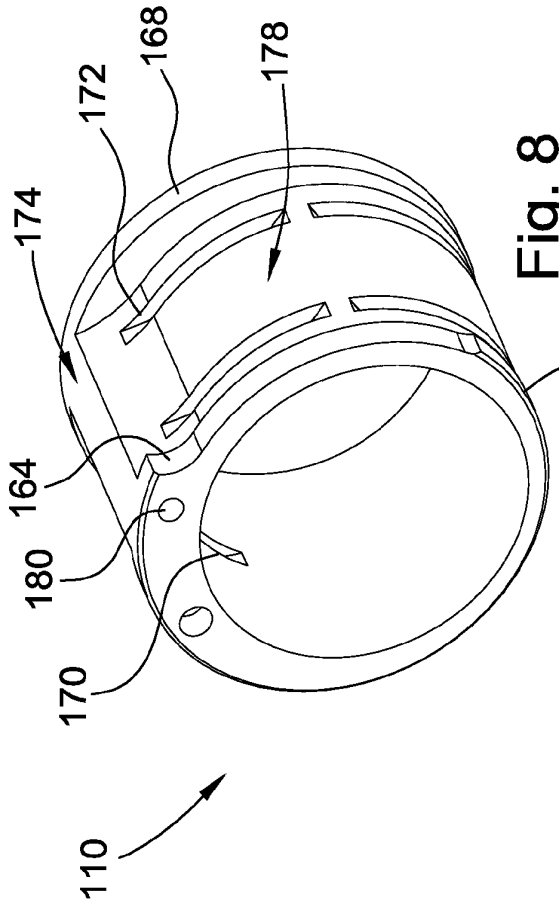

Fig. 8

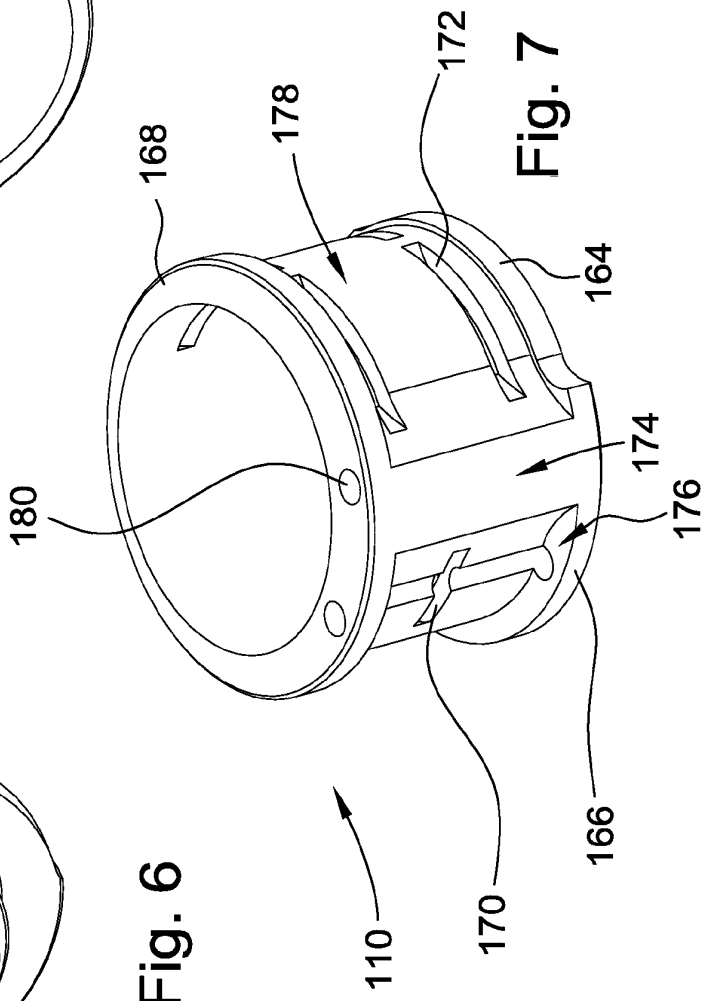

Fig. 7

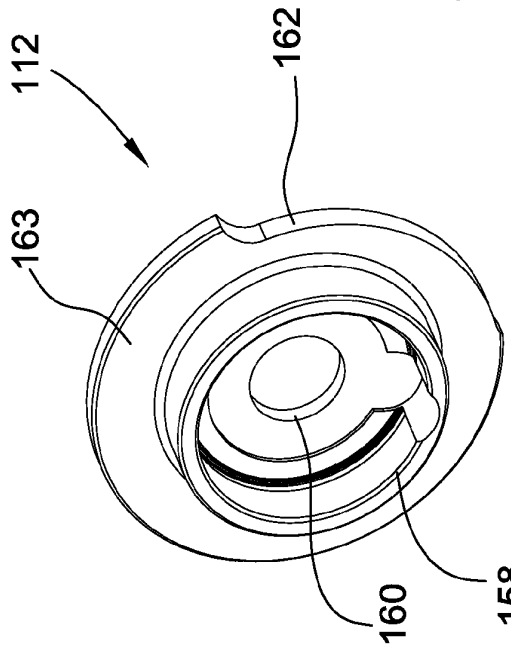

Fig. 6

REFERENCES CITED IN THE DESCRIPTION

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Patent documents cited in the description

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- US 4854085 B, Huber **[0003]**
- EP 0691181 A1 **[0005]**